

## AS Inch/Metric Series

### Selection Guide and Specifications

#### 1. Selection of POWER-LOCK®

a) When only torque is applied:

Compare the max. transmitting torque (T max.) of the devices to be driven with the transmissible torque (Mt) of the POWER-LOCK listed on pages D-21, D-24, and D-25.  
 $Mt \geq T \text{ max} \dots \text{OK}$   
 $Mt < T \text{ max} \dots$  Select a larger POWER-LOCK or use two or more POWER-LOCK units.

The transmissible torque (Mt) of multiple POWER-LOCK units is obtained by multiplying Mt by the number of units used. Peak torque expected should be regarded as T max.

$$T \text{ max.} = \frac{5252 \cdot \text{HP} \cdot \text{s.f.}}{n} \quad (\text{ft./lbs.}) \quad \dots (1)$$

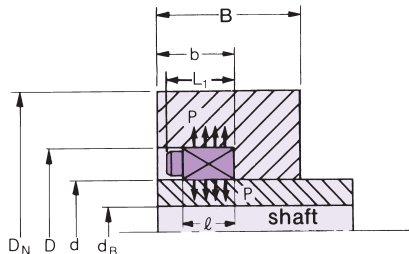
n: RPM    s.f.: Safety Factor

b) When torque and thrust are applied:  
 Compare Mt with composite torque  $M_R$ .

$$M_R = \sqrt{T^2 \text{ max.} + (P \text{ max.} \cdot \frac{d}{2})^2} \leq Mt. \quad \dots (2)$$

T max: Maximum torque (ft./lbs.)  
 P max: Maximum thrust load (lbs.)  
 d: Shaft O.D. (ft.)

#### 2. Hub diameter calculation



a) Standard hub diameter (flange coupling, V sheave, etc.):  
 Min. hub outside diameters ( $D_N$ ) are shown on pages D-22, D-23, D-26 and D-27 based on the yield point of hub material and the length of the hub. Hub diameter must be equal to or larger than ( $D_N$ ). Special hub diameter can be calculated by the following:

b) Hubs of special material:  
 Calculate hub diameter using the following formula (not applicable to non-ferrous metals).

$$Y.P. (\text{hub}) \geq 1.4P'$$

$$Y.P. (\text{shaft}) \geq 1.4P$$

$$\text{Min. hub. dia. } D_N \geq D \sqrt{\frac{Y.P. + K_3 \cdot P'}{Y.P. - K_3 \cdot P'}} \quad \dots (3)$$

$K_3$ : 0.6 ....Using single unit  
 $b \geq L_1, B \geq 2\ell$

$K_3$ : 0.8 ....Using multiple units  
 $b \geq n \cdot L_1$   
 $B \geq n \cdot L_1 + L_1$

$K_3$ : 1.0 ....Using single unit  
 $\ell \leq B < 2 \cdot \ell$   
 Using multiple units  
 $\ell + L_1 \leq B < L_1 (n + 1)$   
 (n: number of POWER-LOCK)

P': Contact pressure on hub (psi)  
 Y.P.: Yield point of hub material (psi)

D: Hub bore inside dia. (in.)

$D_N$ : Min. hub dia. (in.)

c) Hollow Shaft Application:

Inside diameter of the hollow shaft:  $d_B$   
 Compute inside diameter of hollow shaft.

$$d_B \leq d \sqrt{\frac{Y.P. - 2 \cdot P \cdot K_3}{Y.P.}} \quad \dots (4)$$

d: Outside diameter of shaft (in.)

P: Contact pressure on shaft side  
 (Refer to pages D-21, D-24 and D-25) (psi)

$K_3$ : 0.6 ... Using single POWER-LOCK

$K_3$ : 0.8 ... Using multiple POWER-LOCK

#### 3. Hub width

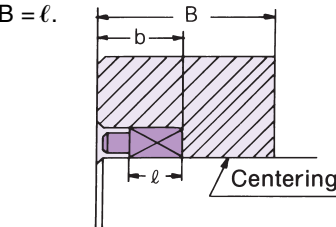
The AS Series POWER-LOCK is not self-aligning. Thus centering ( $B - b$ ) must be performed. Suggested "guide length" ( $B - b$ ) is to be equal to or greater than one half of the shaft diameter.

$B - b \geq d/2$  (d: shaft diameter)

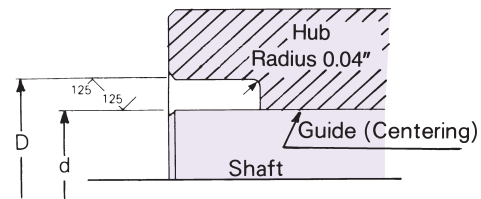
Hub width (B) can be calculated as follows:

$$B \geq b + d/2 \quad \dots (5)$$

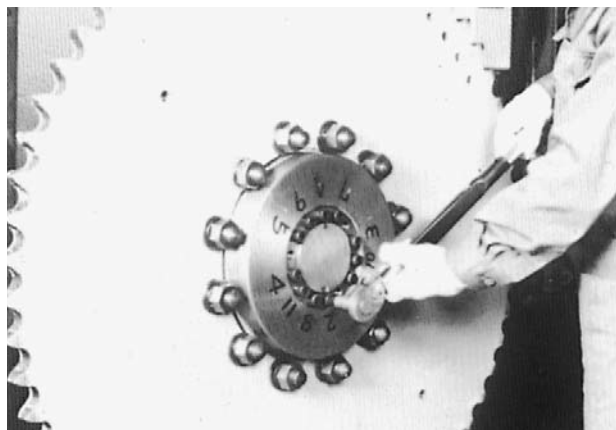
Note: POWER-LOCK cannot be centered under the condition of  $B = \ell$ .



#### 4. Machining tolerance and surface roughness

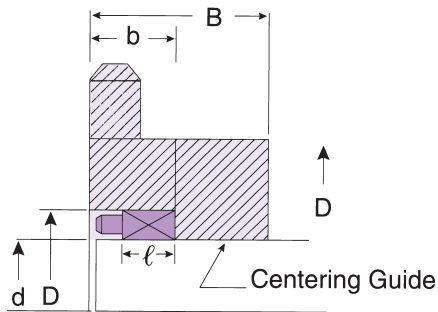


Machining tolerance for counter bore diameter of hub "D" and shaft diameter "d" are listed on pages D-21, D-24 and D-25. Required surface roughness of 125 RMS. Determine the tolerance of the guide according to the centering accuracy required.



### Example Selection 1

A sprocket to be tightened by a POWER-LOCK® on a shaft.



<Conditions>

Shaft Dia.  $d$ : 1½"  
 Max. required torque  $T_{max.}$  = 400 ft./lbs.  
 Sprocket Hub Material 1144 Y.P. = 56,000 psi  
 Sprocket Hub Length ( $B$ ) = 1.875"  
 Sprocket Hub Dia. ( $D_s$ ) = 3.5"  
 Counter Bore Length ( $b$ ) =  $L_1$  (Total PL Length)

#### Step 1

Select PL1½ since shaft dia. is 1½"  
 See Transmissible Torque ( $M_t$ ) on page D-21  
 $M_t (658) \geq T_{max.} (400)$  OK  
 POWER-LOCK Outside Dia.  $D = 2.559$ "  
 POWER-LOCK Total Length  $L_1 = 1.024$ "  
 POWER-LOCK Length  $\ell = 0.709$ "

#### Step 2

Confirm Min. Hub Dia.  $D_N$   
 $2\ell = 2 \cdot 0.709 = 1.418$   $B (1.875) \geq 2\ell (1.418)$  OK  
 Refer Min. Hub Dia.  $D_N$  to page D-22  
 $D_N = 3.030$   
 $D_s (3.5) \geq D_N (3.030)$  OK

#### Step 3

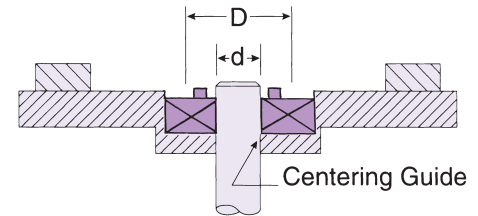
Confirm Sprocket Hub Length  $B$   
 Min. Hub Length  $B_{min.} =$   
 $b + \frac{d}{2} = L_1 + \frac{d}{2} = 1.024 + \frac{1.5}{2} = 1.774$   
 $B (1.875) > B_{min.} (1.774)$  OK  
 Centering Guide will be  $1.875 - b$   
 $= 1.875 - 1.024 = 0.851$   
 $0.851 > \frac{d}{2} (0.75)$  OK

#### Step 4

Machining Tolerance. See page D-21  
 Shaft size ( $d$ ) = 1.5" + 0/-0.0015"  
 Hub Bore Size ( $D$ ) = 2.559" + .0015"/-0

### Example Selection 2

Turn table to be fixed on a vertical straight shaft by a POWER-LOCK.



<Conditions>

Shaft Dia.  $d = 2$ " = 1/6 (ft.)  
 Max. Required Torque  $T_{max.}$  = 500 ft./lbs.  
 Total Table Weight  $P_{max.}$  = 500 ft./lbs.  
 Material for table and hub is 1040

#### Step 1

Select PL2 since shaft Dia. is 2"  
 See Transmissible Torque  $M_t$  and Thrust  $P_{ax}$  on page D-21  
 $M_t = 1627$  ft./lbs.  $P_{ax} = 19,360$  lbs.

#### Step 2

Compare  $M_t$  with composite torque  $MR$ . Calculate composite torque  $MR$ .

$$MR = \sqrt{T^2 \max. + (P_{max.} \cdot \frac{d}{2})^2} = 501.7 \text{ ft./lbs.}$$

$$MR (501.7) \leq M_t (1,627) \text{ OK}$$

#### Step 3

Hub Dimensions. See page D-22  
 Suggested Min. Hub Dia. ( $D_N$ ) = 4.110"  
 Suggested Guide Length =  $\frac{d}{2} = 1$ "

#### Step 4

Machining Tolerance. See page D-21  
 Shaft Size ( $d$ ) = 2" + 0/-0.0018"  
 Hub Bore Size ( $D$ ) = 3.346 + 0.0018"/-0