

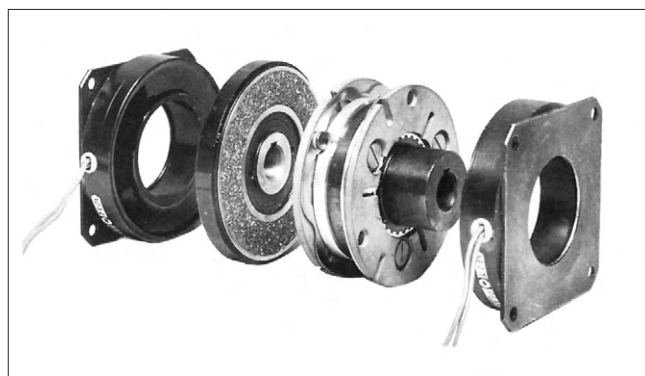
## 1. Specifications

Motor	Output	Three-phase :0.1, 0.2, 0.4, 0.75, 1.5, 2.2, 3.7kW	Single-phase :100W, 200W
	Power supply	200/200/220V 50/60/60Hz	
	Number of poles	4	
	Protection type	0.1 kW - Totally-enclosed type (IP44), 0.2-3.7 kW - Totally-enclosed external fan type (IP44)	Drip-proof protection type (IP22)
	Cooling method	0.1 kW - Self-cooling type (IC410), 0.2-3.7 kW - Self-managed type (IC411)	Draft type (IC01)
	Starting method	-	
	Rating	Continuous	
	Insulation	Class E	
Reducer	Reduction ratio	1/5 to 1/200	
	Speed reducing method	External gear system (helical gear, spur wheel)	
	Lubricating method	Grease lubrication	
	Shaft end key way	New JIS key (JIS B1301-1976): Output shaft key attached	
	Output shaft end	Tapped	
Ambient conditions	Installation place	Indoor not exposed to dust or water	
	Ambient temperature	0°C to 40°C	
	Ambient humidity	Less than 85% (non condensing)	
	Altitude	At elevations below 1000 m	
	Atmosphere	Free from corrosive gases, explosive gases and steam	
	Mounting direction	No limitations on mounting angles: horizontal, vertical or inclined	
Paint color		Munsell 2.5G 6/3	

Note) The protective construction for the CB gear motor is IP12.

## 2. Clutch/Brake

Type	Dry single-plate friction type
Actuation method	Excitation
Rated voltage	24 V DC
Insulation class	Class B
Protection type	Open
Gap adjustment	Automatic gap control system
Lining	No asbestos



For lubrication, installation and coupling, refer to pages 59 to 62.

# CB Gear Motor

GEAR MOTOR TA Series

## Power supply box

The following power supply boxes and control unit for the clutch/brake are available. Make the selection according to your operating conditions. The control unit is of the non-contact type, making it suitable for high frequency on-off switching.

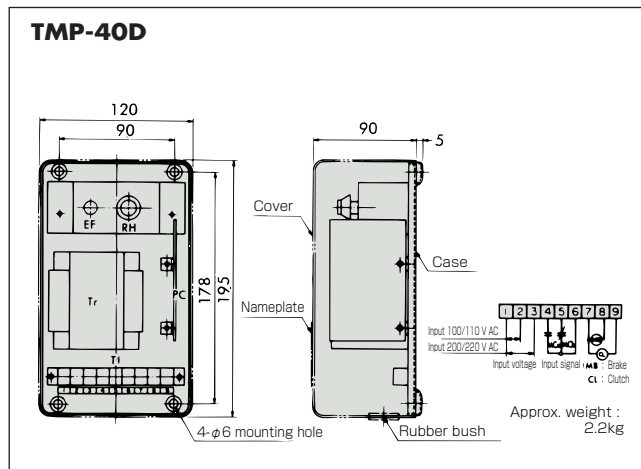
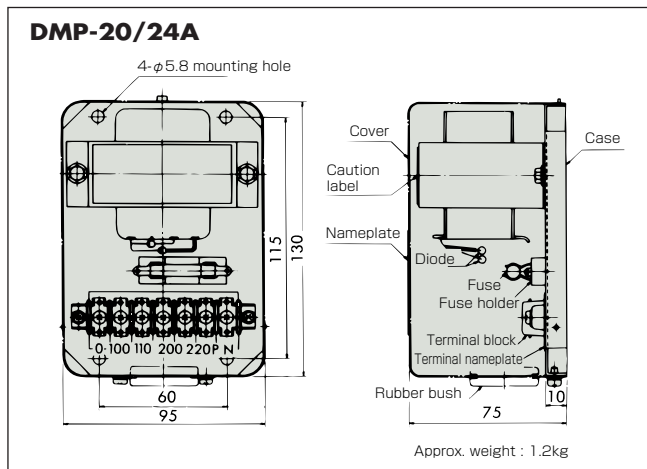
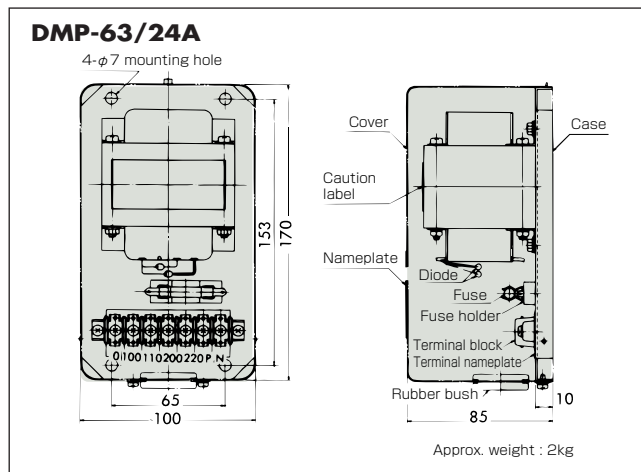
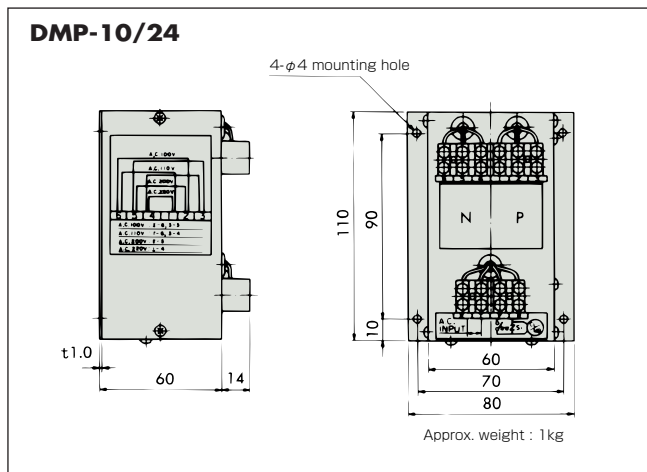


## Power supply box list

Model	Model number	Capacity W	Recommended CB gear motor	Varistor for discharge circuit	Function	Specifications
DMP-type power supply box	DMP-10/24	10	GMTA010 GMTA100 GMTA020 GMTA200 GMTA040	Z15D151	Rectifying only	Input voltage: 100/100/110 V AC 200/200/220 V AC Output voltage: 24 V DC Rating: Continuous Paint color: Munsell 7.5BG6/1.5
	DMP-20/24A	20	GMTA075 GMTA150 GMTA220	Z15D151		
	DMP-63/24A	63	GMTA370	Z15D151		
TMP-type control unit	TMP-40D	40	All models	Unnecessary	Rectifying and brake torque adjusting	

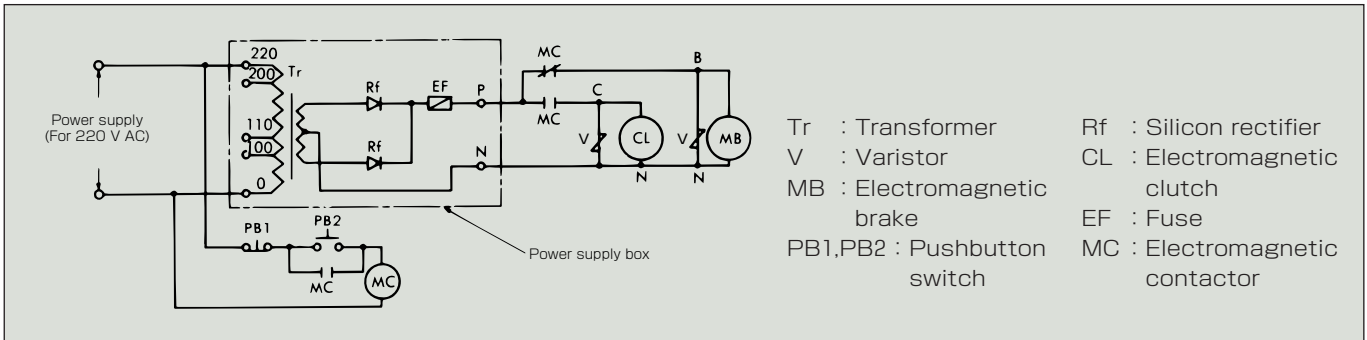
(The varistor accompanies the CB gear motor.)

Specifications



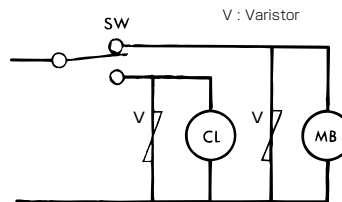
## Control circuit

### 1. Example of circuit using standard power supply box (DMP-type)



### 2. The discharge circuit

When performing switching on the DC side using the standard power supply box (DMP-type), provide a discharge circuit using the varistor accompanying the CB gear motor in order to protect the switch and to prevent a dielectric breakdown of the clutch/brake.



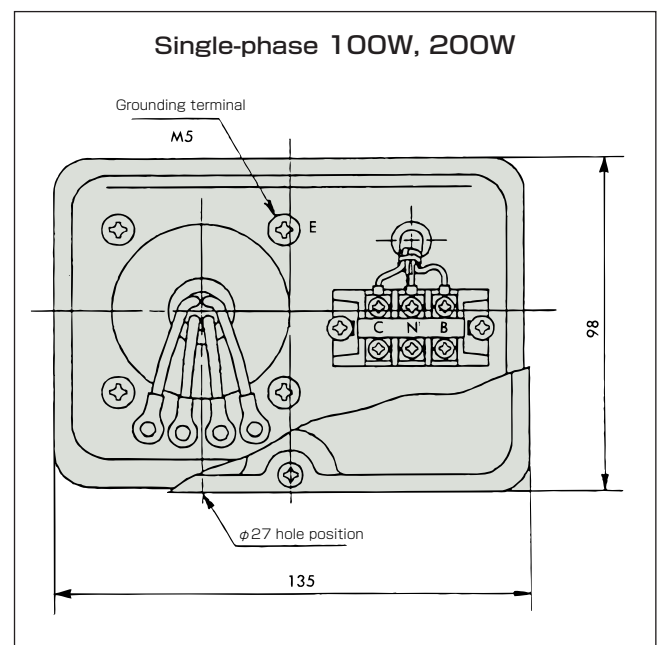
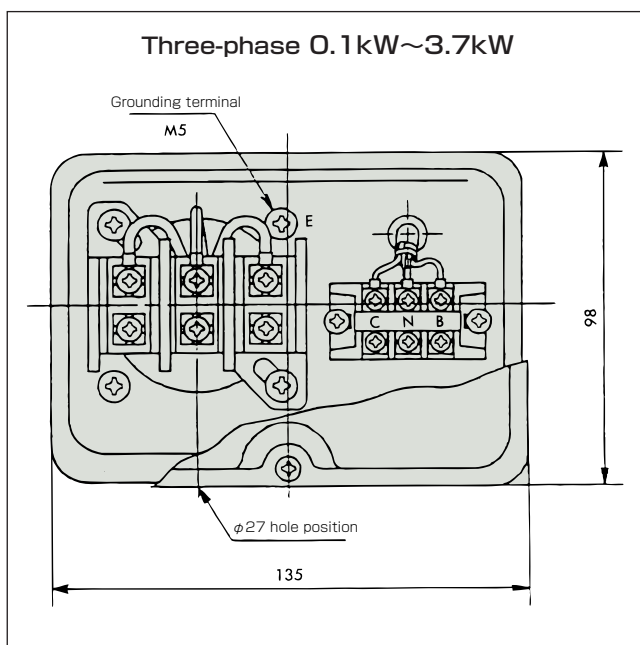
When using the standard power supply box (TMP-type), because a discharge circuit is included, the above procedure is not necessary.

### 3. Power capacity

The power capacity for the electromagnetic clutch/brake should be 130% or more of the power consumption of the clutch/brake.

When using two or more CB gear motors, it should be 130% or more of the total capacity.

### 4. Structure of terminal box

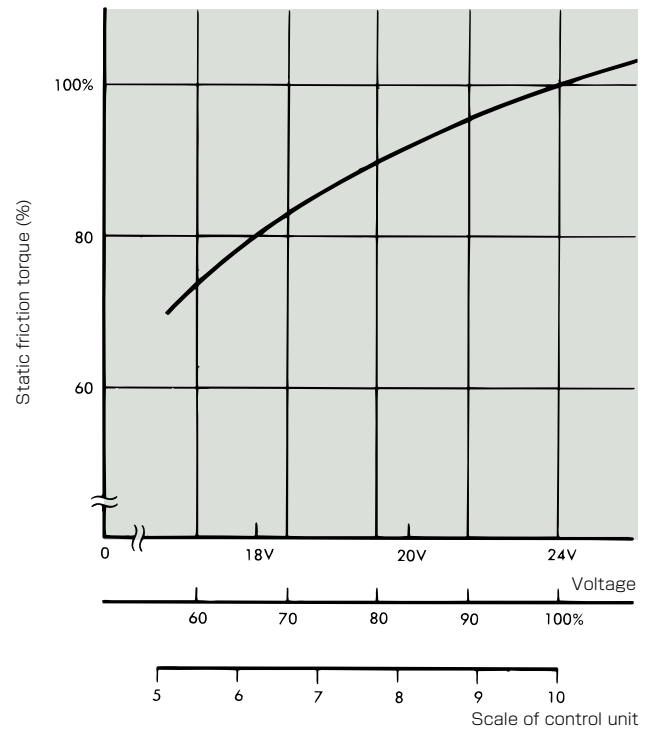


## 5. Brake torque adjusting function

For the TMP-type control unit, a brake torque adjusting resistor is set in the circuit. The brake torque can be adjusted as shown in the figure to the right by reducing the voltage with this resistor. The lower limit of voltage adjustment is about 70%. If the voltage is reduced too much, insufficient pull-in of the armature will result. The scale of the control unit should be regarded as a guideline. To determine the precise value, measure the terminal voltage.

## 6. Special control circuit

If fast actuation of the clutch/brake is required, increase the voltage applied to the clutch/brake above the rated voltage, which will shorten the armature pull-in time and the torque rise time. Using this method will cause heating of the clutch/brake body. In addition, if the  $GD^2$  of the load is 0.5 or more as compared with that of the motor, the shock on starting will be increased. Care should therefore be taken when using this method.



## Selection

### 1. Selection procedure

- When making a selection based on the torque and revolution, select a model number according to the gear motor. The allowable inertia ratio, clutch/brake clutching work and life span are described here.

- 1. Conditions**
- Output shaft revolution speed  $n_L$  and frequency Hz
  - Load torque on motor shaft  $T_e$
  - Load on motor shaft  $GD_i^2$  (moment of inertia)
  - $GD_M^2$  of motor
  - Starting frequency per minute N (Maximum frequency: 60 times/min)
  - Method of coupling to load

- 2. Selection of model** (1) Calculate the corrected inertia ratio (U).

$$U = \frac{GD_i^2 \times U_F}{GD_M^2}$$

Correction coefficient according to coupling method : ( $U_F$ )

Method of coupling to load	$U_F$
Direct coupling, etc., that causes no shake	1.0
Chain transmission, etc., that causes shake	1.5

- (2) Make sure that the corrected inertia ratio (U) of the motor to be used does not exceed the allowable inertia ratio ( $U_{max}$ ).

※If it exceeds the allowable inertia ratio, please contact our company.

- (3) While referring to the clutch/brake selection diagram on page 92, select a suitable motor capacity to allow for the load torque on the motor shaft and  $GD^2$  on motor shaft.

(For N=25, refer to the diagram for N=30.)

When a precise selection is necessary, determine it using the basic formula on page 104.

- 3. Determination of life span** Calculate the clutching work per time  $E_c$  using the formula shown in Section 3 on page 104.

From this  $E_c$  and the total work ( $E_T$  shown on page 105), calculate the total clutching count (Z) as shown below.

$$Z = \frac{E_T}{E_c}$$

To calculate the life span in units of days, use the following formula.

$$Z_d = \frac{Z}{N \times 60 \times N_h}$$

$Z_d$  : Life span in units of days  
 $N$  : Starting count per minute  
 $N_h$  : Average operating time per day

Allowable inertia ratio : ( $U_{max}$ )

Reduction ratio	$U_{max}$
Under 1/30	1.0
1/40~1/50	0.5
1/100~1/200	0.2

## 2. Example of selection

### SI units

#### 1. Conditions

- Output shaft revolution speed  $n_L = 50r/min$  (50Hz)
- Load torque on motor shaft  $T_e = 1.764N \cdot m$
- Moment of inertia on motor shaft  
 $I_e = 0.001kg \cdot m^2$  (Load being coupled directly)
- Moment of inertia of load of motor  $I_M = 0.00119kg \cdot m^2$
- Starting frequency  $N = 15$  times/min

#### 2. Selection of model

$$U = \frac{I_e \times U_F}{I_M} = \frac{0.001 \times 1.0}{0.00119} = 0.84$$

- Because the output shaft revolution speed is 50 r/min, the reduction ratio is 1/30 according to the specification chart (page 106) and therefore the allowable inertia ratio is 1.0, which means there is no problem with the corrected inertia ratio U.
- Because the starting frequency per minute is 15 times/min, refer to the diagram for N = 20 times/min. The point of intersection of the above-shown  $T_e$  and  $I_e$  in this diagram indicates that the proper motor capacity is 0.4 kW. As a result, the model number that should be selected is GMT040-L30CB.

#### 3. Determination of life span

(Perform the following calculation while referring to page 104.)

$$E_c = \frac{\Sigma I \times n^2}{182} \times \frac{T_d}{(T_d - T_e)}$$

$$= \frac{1.216 \times 10^{-3} \times 1500^2}{182} \times \frac{3.53}{(3.53 - 1.76)}$$

$$= 29.9J$$

$$\Sigma I = 2.16 \times 10^{-4} + 1.0 \times 10^{-3}$$

$$= 1.216 \times 10^{-3} kg \cdot m^2$$

$$n = 1500r/min$$

$$T_d = 3.53N \cdot m$$

$$T_e = 1.76N \cdot m$$

$$E_T = 3.92 \times 10^3 J$$

$$Z = \frac{3.92 \times 10^3}{29.9} = 13.1 \times 10^3 \text{ times}$$

### Gravitational units

#### 1. Conditions

- Output shaft revolution speed  $n_L = 50r/min$  (50Hz)
- Load torque on motor shaft  $T_e = 0.18kgf \cdot m$
- Load on motor shaft  $GD_i^2 = 0.004kgf \cdot m^2$  (Load being coupled directly)
- $GD_M^2$  of motor = 0.00476kgf  $\cdot m^2$
- Starting frequency  $N = 15$  times/min

#### 2. Selection of model

$$U = \frac{GD_i^2 \times U_F}{GD_M^2} = \frac{0.004 \times 1.0}{0.00476} = 0.84$$

- Because the output shaft revolution speed is 50 r/min, the reduction ratio is 1/30 according to the specification chart (page 106) and therefore the allowable inertia ratio is 1.0, which means there is no problem with the corrected inertia ratio U.
- Because the starting frequency per minute is 15 times/min, refer to the diagram for N = 20 times/min. The point of intersection of the above-shown  $T_e$  and  $GD_i^2$  in this diagram indicates that the proper motor capacity is 0.4 kW. As a result, the model number that should be selected is GMT040-L30CB.

#### 3. Determination of life span

(Perform the following calculation while referring to page 104.)

$$E_c = \frac{\Sigma GD^2 \times n^2}{7160} \times \frac{T_d}{(T_d - T_e)}$$

$$= \frac{4.86 \times 10^{-3} \times 1500^2}{7160} \times \frac{0.36}{(0.36 - 0.18)}$$

$$= 3.05kgf \cdot m$$

$$\Sigma GD^2 = 8.63 \times 10^{-4} + 4.0 \times 10^{-3}$$

$$= 4.86 \times 10^{-3} kgf \cdot m^2$$

$$n = 1500r/min$$

$$T_d = 0.36kgf \cdot m$$

$$T_e = 0.18kgf \cdot m$$

$$E_T = 4.0 \times 10^7 kgf \cdot m$$

$$Z = \frac{4.0 \times 10^7}{3.05} = 13.1 \times 10^3 \text{ times}$$

## 3. Clutch/Brake selection diagram

### SI units

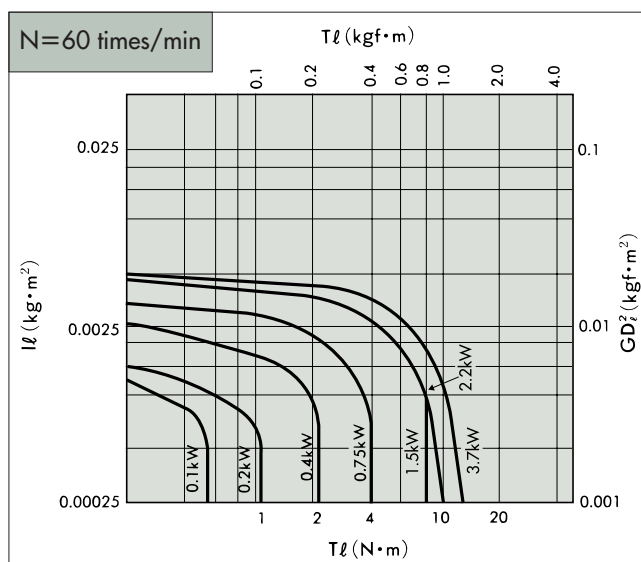
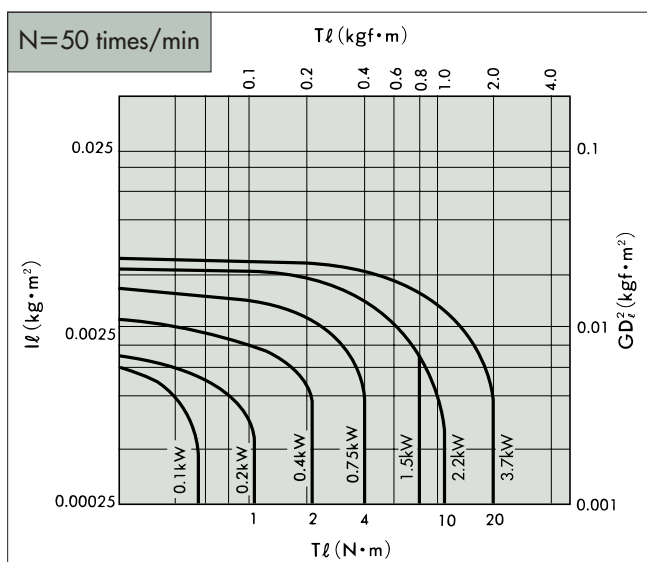
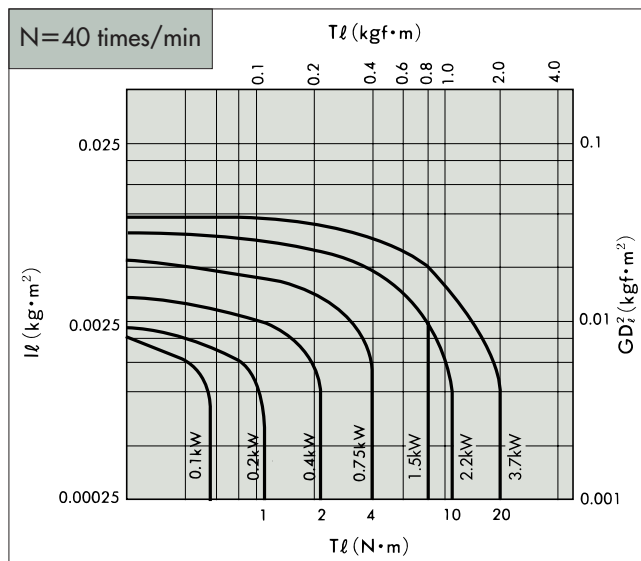
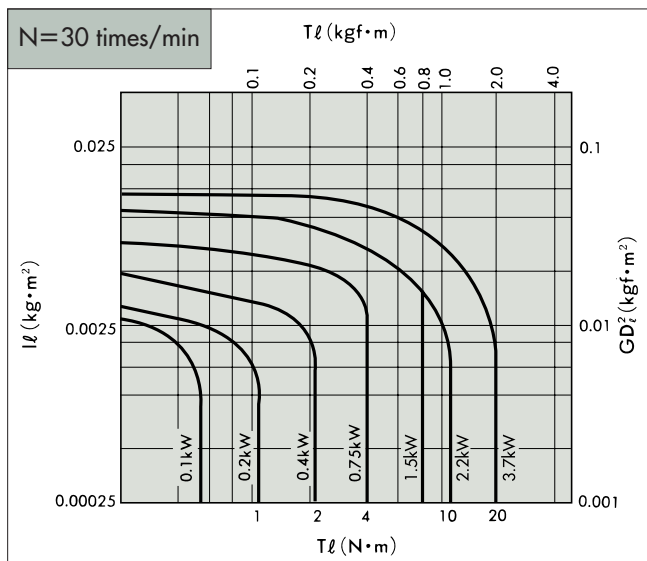
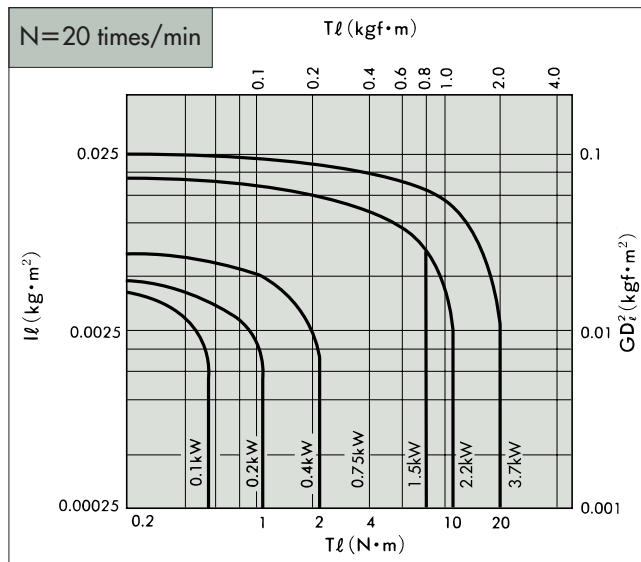
$$T_e = \frac{T_L}{R} \quad I_e = \frac{I_L}{R^2}$$

- $T_e$  : Load torque on motor shaft (N·m)
- $T_L$  : Load torque on output shaft (N·m)
- $I_e$  : Moment of inertia on motor shaft (kg·m<sup>2</sup>)
- $I_L$  : Moment of inertia of load on output shaft (kg·m<sup>2</sup>)
- R : Reduction ratio

### Gravitational units

$$T_e = \frac{T_L}{R} \quad GD_e^2 = \frac{GD_L^2}{R^2}$$

- $T_e$  : Load torque on motor shaft (kgf·m)
- $T_L$  : Load torque on output shaft (kgf·m)
- $GD_e^2$  : Moment of inertia on motor shaft  $GD^2$  (kgf·m<sup>2</sup>)
- $GD_L^2$  : Moment of inertia of load on output shaft  $GD^2$  (kgf·m<sup>2</sup>)
- R : Reduction ratio



## 4. Basic formula for selection

### SI units

#### 1. Torque, revolution speed and power

$$T_L = 9550 \frac{P}{n_L} \quad \begin{array}{l} T_L : \text{Torque on output shaft } \text{N}\cdot\text{m} \\ P : \text{Power } \text{kW} \\ n_L : \text{Output shaft revolution speed } \text{r/min} \end{array}$$

#### 2. Conversion to motor shaft

(1)  $T_e$  : Load torque on motor shaft  $\text{N}\cdot\text{m}$

$$T_e = \frac{T_L}{i} \quad \begin{array}{l} T_L : \text{Torque on output shaft } \text{N}\cdot\text{m} \\ i : \text{Reduction ratio} \end{array}$$

(2)  $I_e$  : Moment of inertia on motor shaft  $\text{kg}\cdot\text{m}^2$

$$I_e = \frac{I_L}{i^2}$$

$I_L$  : Moment of inertia on output shaft  $\text{kg}\cdot\text{m}^2$

#### 3. Clutching (braking) work

(1)  $E_e$  : Clutching (braking) work per time  $\text{J}$

$$E_e = \frac{\Sigma I \times n_e^2}{182.5} \times \frac{T_d}{(T_d \pm T_e)}$$

$$\Sigma I = I_c + I_e$$

$I_c$  : Moment of inertia of clutch/brake  
 (Table 1 on page 105)

$n_e$  : Motor shaft revolution speed  $\text{r/min}$

$T_d$  : Dynamic friction torque of clutch/brake  $\text{N}\cdot\text{m}$   
 (Table 3 on page 105)

+ : Positive torque at the time of braking

- : Negative torque at the time of clutching and braking

(2)  $E_N$  : Clutching (braking) work per minute  $\text{J}$

$$E_N = E_e \times N$$

$N$  : Starting frequency (times/min)

$$E_N \leq E_o$$

$E_o$  : Allowable work  $\text{J/min}$  (Table 3 on page 105)

#### 4. Clutching (braking) time

$t_b$  : Clutching (braking) time  $\text{s}$

$$t_b = \frac{\Sigma I}{9.55} \times \frac{n_e}{(T_d \pm T_e)}$$

+ : Positive torque at the time of braking

- : Negative torque at the time of clutching and braking

#### 5. Response time

$t_a$  : Armature pull-in time  $\text{s}$  (Table 2 on page 105)

#### 6. Calculation of braking distance

$$S = (t_a + \frac{1}{2}t_b) \times V$$

$S$  : Braking distance  $\text{mm}$

$V$  : Speed (conveyor speed, etc.)  $\text{mm/s}$

#### 7. Stopping accuracy

Precise determination of the stopping accuracy is difficult because it is affected by voltage, temperature, alteration through use, operating time, etc. However, if the following variations are assumed:

- a. Load torque variation:  $\pm 20\%$
- b. Braking torque variation:  $\pm 20\%$
- c. Time lag variation:  $\pm 0.01\text{s}$

The variation from the calculated braking distance is about  $\pm 30\%$ .

Stopping accuracy  $\delta = S \times 0.6$  or  $S \times \pm 0.3$   
 For example, when the calculated braking distance  $S$  is 10 mm, the stopping accuracy is 6 mm ( $10 \pm 3$  mm).

### Gravitational units

#### 1. Torque, revolution speed and power

$$T_L = 974 \frac{P}{n_L} \quad \begin{array}{l} T_L : \text{Torque on output shaft } \text{kgf}\cdot\text{m} \\ P : \text{Power } \text{kW} \\ n_L : \text{Output shaft revolution speed } \text{r/min} \end{array}$$

#### 2. Conversion to motor shaft

(1)  $T_e$  : Load torque on motor shaft  $\text{kgf}\cdot\text{m}$

$$T_e = \frac{T_L}{i} \quad \begin{array}{l} T_L : \text{Torque on output shaft } \text{kgf}\cdot\text{m} \\ i : \text{Reduction ratio} \end{array}$$

(2)  $GD_e^2$  : Load on motor shaft  $\text{GD}^2$   $\text{kgf}\cdot\text{m}^2$

$$GD_e^2 = \frac{GD_L^2}{i^2}$$

$GD_L^2$  : Load on output shaft  $\text{GD}^2$   $\text{kgf}\cdot\text{m}^2$

#### 3. Clutching (braking) work

(1)  $E_e$  : Clutching (braking) work per time  $\text{kgf}\cdot\text{m}$

$$E_e = \frac{\Sigma GD^2 \times n_e^2}{7160} \times \frac{T_d}{(T_d \pm T_e)}$$

$$\Sigma GD^2 = GD_c^2 + GD_e^2$$

$GD_c^2$  : Load on motor shaft  $\text{GD}^2$   
 $GD_e^2$  :  $\text{GD}^2$  of clutch/brake  
 (Table 1 on page 105)

$n_e$  : Motor shaft revolution speed  $\text{r/min}$

$T_d$  : Dynamic friction torque of clutch/brake  $\text{kgf}\cdot\text{m}$   
 (Table 3 on page 105)

+ : Positive torque at the time of braking

- : Negative torque at the time of clutching and braking

(2)  $E_N$  : Clutching (braking) work per minute  $\text{kgf}\cdot\text{m/min}$

$$E_N = E_e \times N$$

$N$  : Starting frequency (times/min)

$$E_N \leq E_o$$

$E_o$  : Allowable work  $\text{kgf}\cdot\text{m/min}$  (Table 3 on page 105)

#### 4. Clutching (braking) time

$t_b$  : Clutching (braking) time  $\text{s}$

$$t_b = \frac{\Sigma GD^2}{375} \times \frac{n_e}{(T_d \pm T_e)}$$

+ : Positive torque at the time of braking

- : Negative torque at the time of clutching and braking

#### 5. Response time

$t_a$  : Armature pull-in time  $\text{s}$  (Table 2 on page 105)

#### 6. Calculation of braking distance

$$S = (t_a + \frac{1}{2}t_b) \times V$$

$S$  : Braking distance  $\text{mm}$

$V$  : Speed (conveyor speed, etc.)  $\text{mm/s}$

#### 7. Stopping accuracy

Precise determination of the stopping accuracy is difficult because it is affected by voltage, temperature, alteration through use, operating time, etc. However, if the following variations are assumed:

- a. Load torque variation:  $\pm 20\%$
- b. Braking torque variation:  $\pm 20\%$
- c. Time lag variation:  $\pm 0.01\text{s}$

The variation from the calculated braking distance is about  $\pm 30\%$ .

Stopping accuracy  $\delta = S \times 0.6$  or  $S \times \pm 0.3$   
 For example, when the calculated braking distance  $S$  is 10 mm, the stopping accuracy is 6 mm ( $10 \pm 3$  mm).



# CB Gear Motor Specification Chart

GEAR MOTOR TA Series

## Specifications of motor and clutch/brake

Table 1. Moment of inertia of motor and clutch/brake

Motor output	Moment of inertia of motor		Model number of clutch/brake		Moment of inertia of clutch/brake	
	$I_M$	$GD_M^2$	Clutch	Brake	$I_c$	$GD_c^2$
	$\text{kg}\cdot\text{m}^2$	$\{\text{kgf}\cdot\text{m}^2\}$			$\text{kg}\cdot\text{m}^2$	$\{\text{kgf}\cdot\text{m}^2\}$
0.1kW	$0.64 \times 10^{-3}$	$2.54 \times 10^{-3}$	NC/NB-0.15/0.1-AG-001		$1.21 \times 10^{-4}$	$\{4.83 \times 10^{-4}\}$
100W	$0.60 \times 10^{-3}$	$2.40 \times 10^{-3}$				
0.2kW	$0.74 \times 10^{-3}$	$2.96 \times 10^{-3}$	NC/NB-0.3/0.2-AG-001		$1.21 \times 10^{-4}$	$\{4.83 \times 10^{-4}\}$
200W	$0.88 \times 10^{-3}$	$3.50 \times 10^{-3}$				
0.4kW	$0.90 \times 10^{-3}$	$3.59 \times 10^{-3}$	NC-0.6AG-033	NB-0.4AG-001	$2.16 \times 10^{-4}$	$\{8.63 \times 10^{-4}\}$
0.75kW	$1.37 \times 10^{-3}$	$5.48 \times 10^{-3}$	NC-1.2AG-034	NB-0.75AG-001	$0.62 \times 10^{-3}$	$\{2.46 \times 10^{-3}\}$
1.5kW	$3.41 \times 10^{-3}$	$13.6 \times 10^{-3}$	NC-2.5AG-033	NB-1.5AG-001	$1.94 \times 10^{-3}$	$\{7.74 \times 10^{-3}\}$
2.2kW	$4.79 \times 10^{-3}$	$19.2 \times 10^{-3}$	NC-2.5AG-033	NB-2.2AG-001	$1.94 \times 10^{-3}$	$\{7.74 \times 10^{-3}\}$
3.7kW	$7.60 \times 10^{-3}$	$30.4 \times 10^{-3}$	NC-5AG-024	NB-3.7AG-001	$0.49 \times 10^{-2}$	$\{1.94 \times 10^{-2}\}$

Table 2. Armature pull-in time

	Armature pull-in time (S)	Model number of power supply box
NC/NB-0.15/0.1	0.010	DMP
	0.015	TMP
	0.005	EMP
NC/NB-0.3/0.2	0.010	DMP
	0.015	TMP
	0.005	EMP
NC/NB-0.6/0.4	0.015	DMP
	0.023	TMP
	0.008	EMP
NC/NB-1.2/0.75	0.020	DMP
	0.030	TMP
	0.010	EMP

	Armature pull-in time (S)	Model number of power supply box
NC/NB-2.5/1.5	0.030	DMP
	0.045	TMP
	0.015	EMP
NC/NB-2.5/2.2	0.035	DMP
	0.053	TMP
	0.018	EMP
NC/NB-5/3.7	0.040	DMP
	0.060	TMP
	0.020	EMP

EMP has special specifications for double excitation.

Table 3. Specifications of clutch/brake

Motor output	Classification	Model number of clutch/brake	Static friction torque		Dynamic friction torque		Power consumption (W)	Allowable work $E_o$		Total work $E_r$	
			N·m	{kgf·m}	N·m	{kgf·m}		J/min	{kgf·m/min}	J	{kgf·m}
0.1kW/100W	Clutch	NC/NB-0.15/0.1-AG-001	1.47	{0.15}	0.88	{0.09}	4	1960	{200}	$25.5 \times 10^7$	$\{2.6 \times 10^7\}$
	Brake		0.98	{0.1}	0.59	{0.06}	3	1764	{180}		
0.2kW/200W	Clutch	NC/NB-0.3/0.2-AG-001	2.94	{0.3}	1.76	{0.18}	5	1960	{200}	$25.5 \times 10^7$	$\{2.6 \times 10^7\}$
	Brake		1.96	{0.2}	1.18	{0.12}	4	1764	{180}		
0.4kW	Clutch	NC-0.6AG-033	5.88	{0.6}	3.53	{0.36}	8	2744	{280}	$39.2 \times 10^7$	$\{4.0 \times 10^7\}$
	Brake	NB-0.4AG-001	3.92	{0.4}	2.35	{0.24}	7	2450	{250}		
0.75kW	Clutch	NC-1.2AG-034	11.8	{1.2}	6.86	{0.70}	11	4410	{450}	$61.7 \times 10^7$	$\{6.3 \times 10^7\}$
	Brake	NB-0.75AG-001	7.35	{0.75}	4.41	{0.45}	8	3969	{405}		
1.5kW	Clutch	NC-2.5AG-033	24.5	{2.5}	14.7	{1.5}	17	6860	{700}	$90.2 \times 10^7$	$\{9.2 \times 10^7\}$
	Brake	NB-1.5AG-001	14.7	{1.5}	8.82	{0.9}	12	6174	{630}		
2.2kW	Clutch	NC-2.5AG-033	24.5	{2.5}	14.7	{1.5}	17	6860	{700}	$90.2 \times 10^7$	$\{9.2 \times 10^7\}$
	Brake	NB-2.2AG-001	21.6	{2.2}	12.7	{1.3}	16	6174	{630}		
3.7kW	Clutch	NC-5AG-024	49.0	{5.0}	29.4	{3.0}	25	10290	{1050}	$16.7 \times 10^8$	$\{1.7 \times 10^8\}$
	Brake	NB-3.7AG-001	36.3	{3.7}	22.5	{2.3}	17	9310	{950}		

## Precautions for use

### ●Use with constantly held loads

Because the brake of the CB gear motor exerts a braking effect through excitation, it is unsuitable for use in equipment, such as hoisting machines, that always holds loads.

### ●Provision of totally-enclosed types

The standard product cannot be used in an outdoor place where it will be exposed to water or an indoor place where there is lots of dust or oil mist, because the clutch/brake part is of open type. For such uses, a totally-enclosed type clutch/brake is required. For more information, contact our company.

### ●Handling variable voltage and double voltage

For a motor, it is possible to cope with variable voltage and double voltage at time of purchase or through rewinding. For the power supply box for a clutch/brake, because it is for 100/200 V AC, a 100 V instrumentation power supply should be used.

### ●Explosion-proof type

No explosion-proof-type clutch/brake is available.